

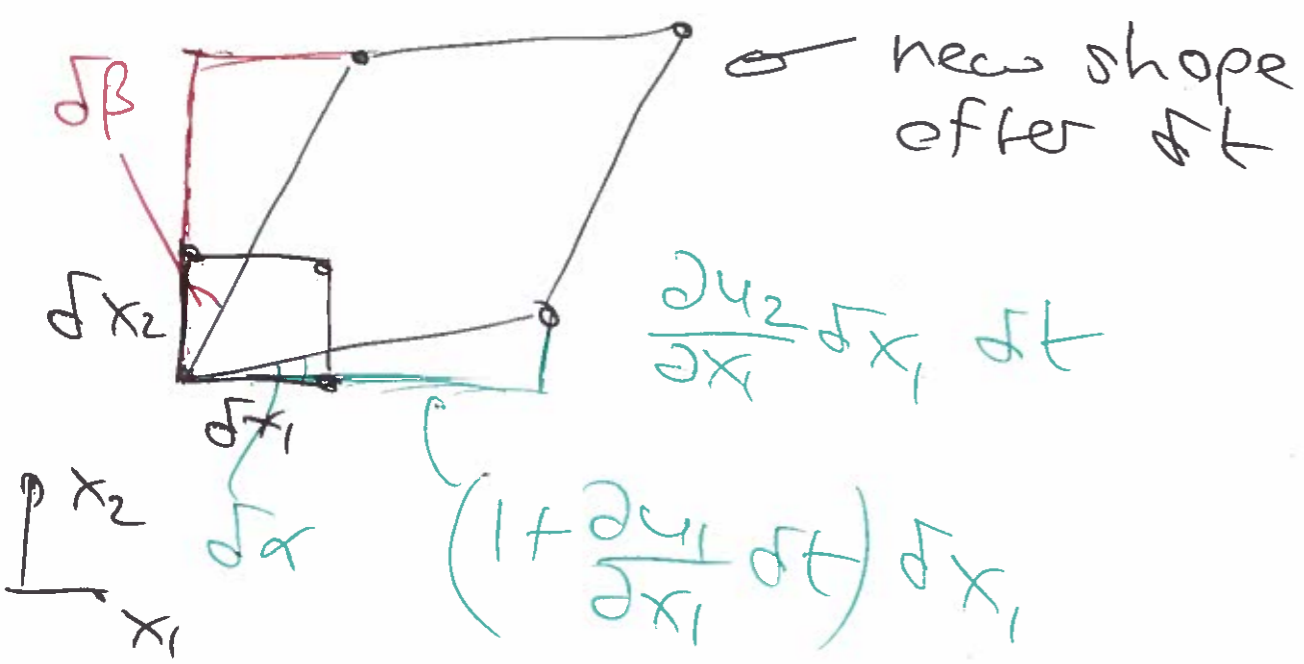
$$\delta u_i = \underbrace{\epsilon_{ij} \delta x_j}_{\substack{\text{dilation} \\ \text{(stretching) /} \\ \& \text{shearing}}} + \underbrace{\omega_{ij} \delta x_j}_{\substack{\text{rigid body} \\ \text{rotation}}}$$

$\epsilon_{ij}$  = rate of strain of material line in  $x_i$ -dir.

(ii) Shear rate of strain

(2D sketch)

2.



$$\tan \delta \alpha = \frac{\frac{\partial u_2}{\partial x_1} \delta x_1 dt}{(1 + \frac{\partial u_1}{\partial x_1} dt) \delta x_1}$$

$\delta \alpha$

$\rightarrow 0$  as  $dt \rightarrow 0$

$$\delta \alpha = \frac{\partial u_2}{\partial x_1} dt$$

$$\frac{D\alpha}{Dt} = \frac{\partial u_2}{\partial x_1}$$

rate of change of angle  $\alpha$  experienced by material line

Similarly

3.

$$\frac{D\beta}{Dt} = \frac{\partial u_1}{\partial x_2} \quad (\text{EXERCISE})$$

Now consider the "shear rate" i.e. the rate of which the angle between these lines changes:

$$d\gamma = d\alpha + d\beta$$



$$\begin{aligned} \frac{D\gamma}{Dt} &= \frac{D\alpha}{Dt} + \frac{D\beta}{Dt} = \left( \frac{\partial u_2}{\partial x_1} + \frac{\partial u_1}{\partial x_2} \right) \\ &= 2\varepsilon_{12} \end{aligned}$$

The off-diagonal entries in the rate of strain tensor  $\varepsilon_{ij}$  represent half the shear rate in the plane spanned by  $x_i$  &  $x_j$ .

Also: The change in angle  $\angle$   
of the main diagonal in  
the fluid rectangle is



~~$$d\theta = \frac{1}{2} \frac{d\alpha - d\beta}{dt}$$~~

$$d\theta = \frac{1}{2} (d\alpha - d\beta)$$

$$\frac{d\theta}{dt} = \frac{1}{2} \left( \frac{d\alpha}{dt} - \frac{d\beta}{dt} \right)$$

$$= \frac{1}{2} \left( \frac{\partial u_2}{\partial x_1} - \frac{\partial u_1}{\partial x_2} \right) = \omega_{21} = \omega_3$$

from rate of  
rotation tensor.

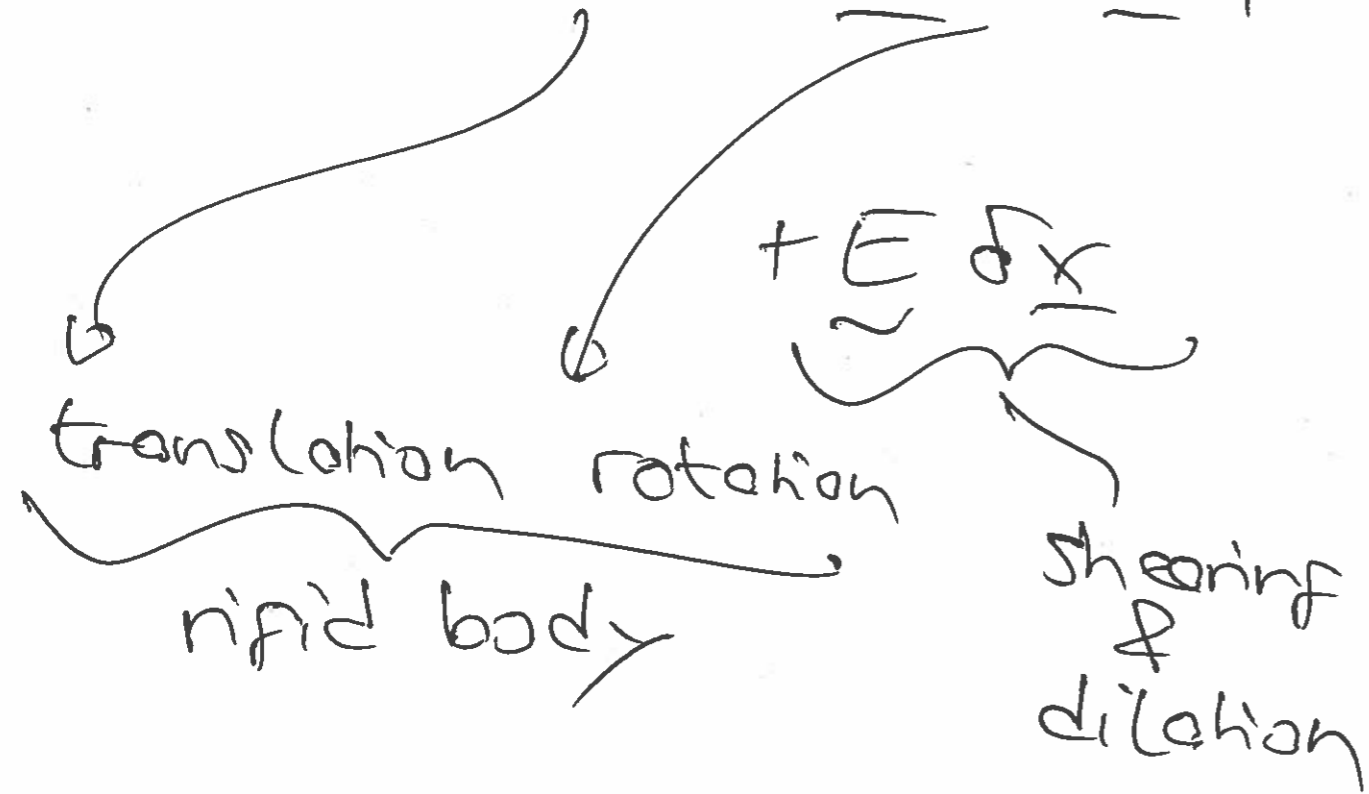
$$\underline{\omega} = \begin{pmatrix} \omega_1 \\ \omega_2 \\ \omega_3 \end{pmatrix}$$

Summary:

(5)

Motion of fluid in the vicinity of a spatial point can be decomposed into

$$\underline{u}(\underline{x} + \underline{\delta x}) = \underline{u}(\underline{x}) + \underline{\Omega} \times \underline{\delta x} +$$



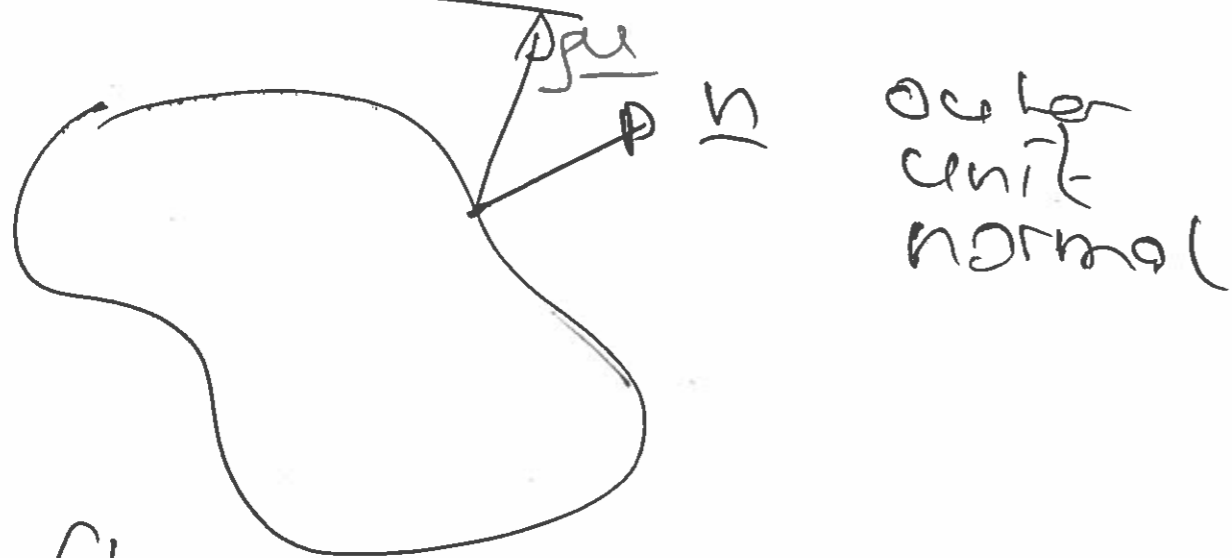
$$u_i(x_j + \delta x_j) = u_i(x_j) + \omega_{ij} \delta x_j + \epsilon_{ij} \delta x_j$$

# Eqn. of continuity

6

Physics: Mass flux into a spatially fixed (control) volume is equal to the rate of change of mass in that volume.

Integral form:



Mass Flux:

density  $\rho$   $\left[ \frac{\text{kg}}{\text{m}^3} \right]$   $\times$  velocity  
normal to boundary  $\left[ \frac{\text{m}}{\text{sec}} \right]$   $\times$   
surface area  $\left[ \text{m}^2 \right]$

$$-\oint \rho \underline{u} \cdot \underline{n} \, dA = \int \frac{d\rho}{dt} \, dV \quad (7)$$

$$\int \frac{d\rho}{dt} \, dV + \oint (\rho \underline{u}) \cdot \underline{n} \, dA = 0$$

$$\int \frac{d\rho}{dt} \, dV + \int \nabla \cdot (\rho \underline{u}) \, dV = 0$$

↓ divergence theorem

$$\int \left( \frac{d\rho}{dt} + \nabla \cdot (\rho \underline{u}) \right) \, dV = 0$$

if this has to hold for all control volumes the integrand has to vanish everywhere:

$$\boxed{\frac{d\rho}{dt} + \nabla \cdot (\rho \underline{u}) = 0}$$