

$$\underline{u} = u_r \underline{e}_r + u_\varphi \underline{e}_\varphi + u_z \underline{e}_z$$

$$u_r = u$$

$$u_\varphi = v$$

$$u_z = w$$

To illustrate the origin of some of the terms in cyl. polars consider rigid body rotation:

$$\underline{u} = u_\varphi \underline{e}_\varphi$$

$$u_\varphi = \Omega r = v$$

$$u, w = 0$$

radial mom. eqn:

$$+\frac{v^2}{r} = +\frac{1}{\rho} \frac{dp}{dr}$$

$$\rho \frac{\Omega^2 r^2}{r} = \frac{dp}{dr}$$

$$\frac{dp}{dr} = \rho \Omega^2 r$$

$$\cancel{\frac{\partial u}{\partial t}} + u \cancel{\frac{\partial u}{\partial r}} + \frac{v \partial u}{r \partial \varphi} + w \cancel{\frac{\partial u}{\partial z}} - \frac{v^2}{r} = -\frac{1}{\rho} \frac{\partial P}{\partial r} + \nu \left[\cancel{\nabla^2 u} - \frac{u}{r^2} - \frac{2}{r^2} \frac{\partial v}{\partial \varphi} \right],$$

$$\frac{\partial v}{\partial t} + u \frac{\partial v}{\partial r} + \frac{v \partial v}{r \partial \varphi} + w \frac{\partial v}{\partial z} + \frac{uv}{r} = -\frac{1}{\rho r} \frac{\partial P}{\partial \varphi} + \nu \left[\nabla^2 v - \frac{v}{r^2} + \frac{2}{r^2} \frac{\partial u}{\partial \varphi} \right],$$

$$\frac{\partial w}{\partial t} + u \frac{\partial w}{\partial r} + \frac{v \partial w}{r \partial \varphi} + w \frac{\partial w}{\partial z} = -\frac{1}{\rho} \frac{\partial P}{\partial z} + \nu \nabla^2 w,$$

$$\operatorname{div} \mathbf{u} = \frac{1}{r} \frac{\partial}{\partial r} (ru) + \frac{1}{r} \frac{\partial v}{\partial \varphi} + \frac{\partial w}{\partial z} = 0.$$

where

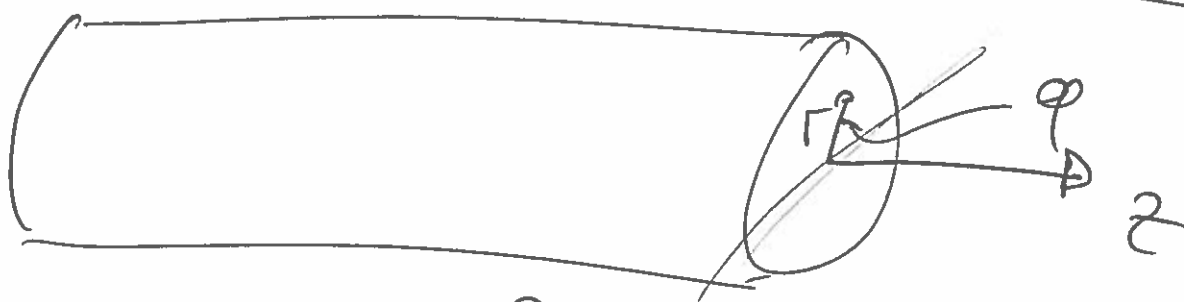
$$\nabla^2 \equiv \frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2}{\partial \varphi^2} + \frac{\partial^2}{\partial z^2}.$$

$$p = \frac{1}{2} \rho \omega^2 r^2 + p_0$$

↑
const.

This ~~is~~ is due to centrifugal
"forces"

Example: Hagen-Poiseuille flow



pipe radius R .

Assume:

$$\underline{u} = u_z \underline{e}_z =$$

$$= \underbrace{u_z(r)}_{\omega(r)} \underline{e}_z$$

$$p = p(r, z)$$

$$U = v = 0$$

$$\omega (\mathbb{R})$$

$$P(\mathbb{R}, z)$$

$$\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial r} + \frac{v \partial u}{r \partial \varphi} + w \frac{\partial u}{\partial z} - \frac{v^2}{r} = -\frac{1}{\rho} \frac{\partial P}{\partial r} + \nu \left[\nabla^2 u - \frac{u}{r^2} - \frac{2}{r^2} \frac{\partial v}{\partial \varphi} \right],$$

$$\frac{\partial v}{\partial t} + u \frac{\partial v}{\partial r} + \frac{v \partial v}{r \partial \varphi} + w \frac{\partial v}{\partial z} + \frac{uv}{r} = -\frac{1}{\rho r} \frac{\partial P}{\partial \varphi} + \nu \left[\nabla^2 v - \frac{v}{r^2} + \frac{2}{r^2} \frac{\partial u}{\partial \varphi} \right],$$

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where

$$\nabla^2 \equiv \frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2}{\partial \varphi^2} + \frac{\partial^2}{\partial z^2}.$$

z - comp:

$\omega(r)$
 $\rho(z)$

3

$$0 = \underbrace{-\frac{1}{5} \frac{\partial p}{\partial z}}_{\text{fcf of } z} + \mu \underbrace{\left(\frac{\partial^2 \omega}{\partial r^2} + \frac{1}{r} \frac{\partial \omega}{\partial r} \right)}_{\text{fcf of } r}$$

fcf of z
(at most)

fcf of r
(at most)

As before

$$\frac{\partial p}{\partial z} = \text{const} = G$$

$$\frac{G}{\mu} = \underbrace{\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial \omega}{\partial r} \right)}_{\nabla^2 \omega}$$

$$\frac{G}{\mu} r = \frac{\partial}{\partial r} \left(r \frac{\partial \omega}{\partial r} \right)$$

$$A + \frac{1}{2} \frac{G}{\mu} r^2 = r \frac{\partial \omega}{\partial r}$$

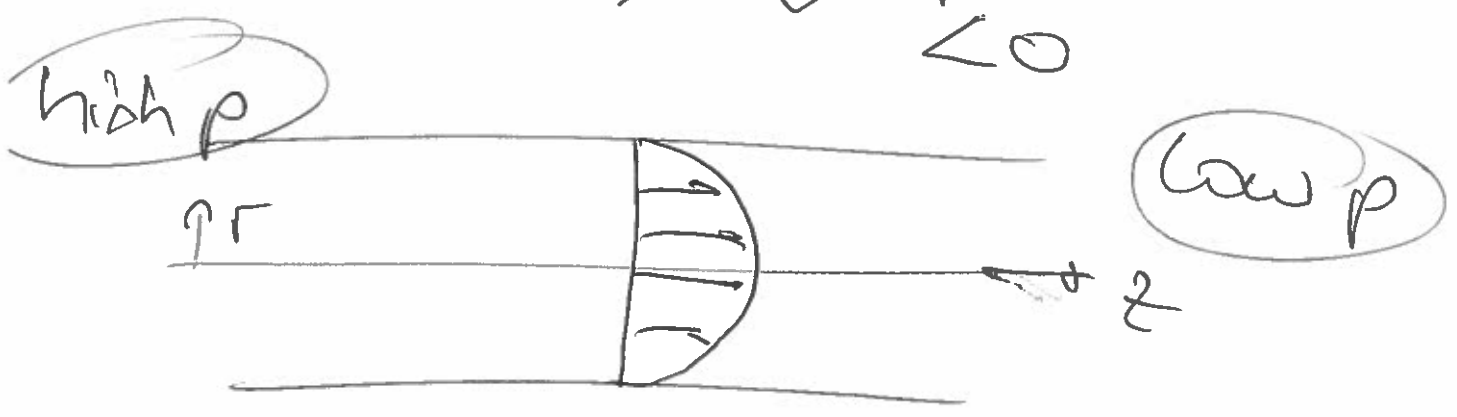
$$\frac{\partial \omega}{\partial r} = \frac{1}{2} \frac{G}{\mu} r + \frac{A}{r}$$

$$\omega(r) = \frac{1}{4} \frac{G}{\mu} r^2 + A \ln r + B$$

A, B from BCs:

$\omega(r=R) = 0$ (no slip)
 $\omega(r=0)$ finite: $A=0$

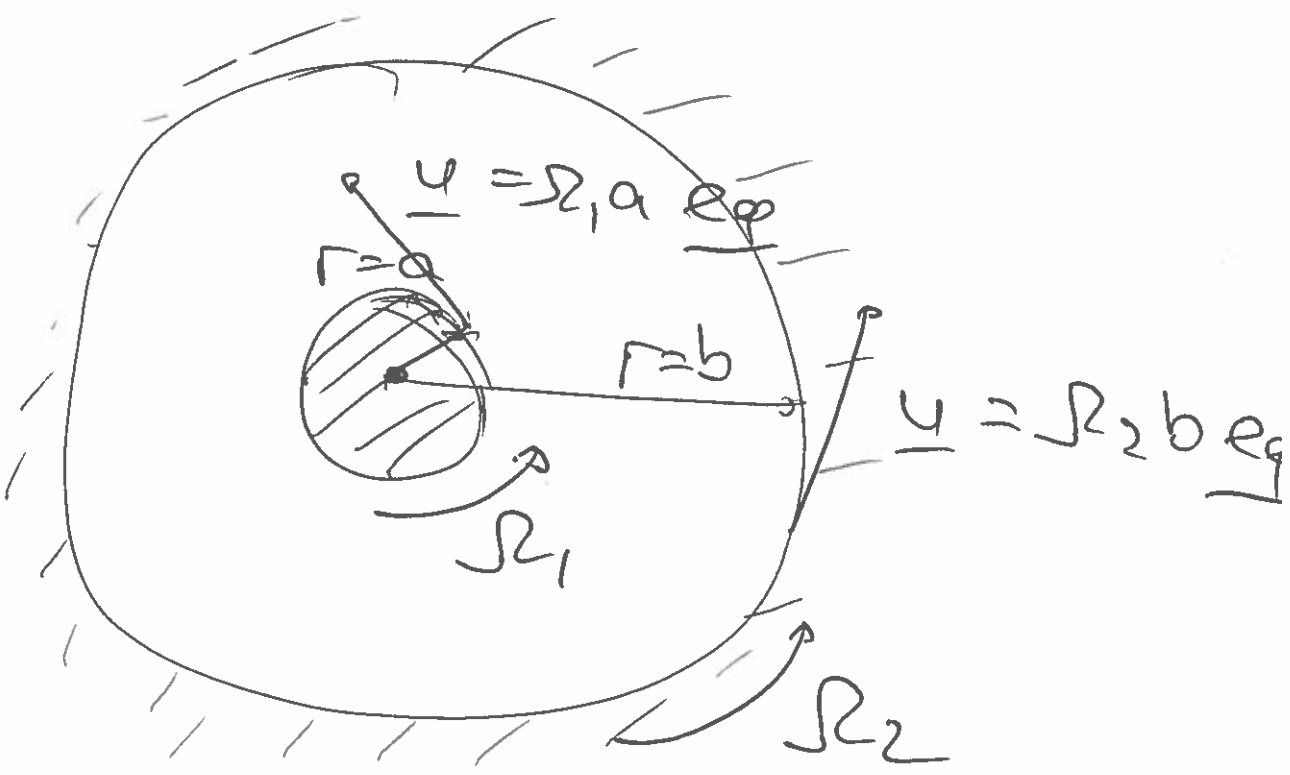
$$\omega(r) = \frac{1}{4} \frac{G}{\mu} (r^2 - R^2)$$



$G = \frac{dp}{dz} < 0$ for sketch to be correct



Example: Circular Couette flow



Assumptions

- Steady
 - $\underline{u} = u_\phi \underline{e}_\phi$
 - $\frac{\partial}{\partial \phi} = 0$
 - $\frac{\partial}{\partial z} = 0$
- } $\underline{u} = u_\phi(r) \underline{e}_\phi$

Flow driven by wall (Couette)

$\nabla p = 0$

$$\frac{\partial u}{\partial t} + u \frac{\partial u}{\partial r} + \frac{v}{r} \frac{\partial u}{\partial \varphi} + w \frac{\partial u}{\partial z} - \frac{v^2}{r} = -\frac{1}{\rho} \frac{\partial P}{\partial r} + \nu \left[\nabla^2 u - \frac{u}{r^2} - \frac{2}{r^2} \frac{\partial v}{\partial \varphi} \right],$$

~~$$\frac{\partial v}{\partial t} + u \frac{\partial v}{\partial r} + \frac{v}{r} \frac{\partial v}{\partial \varphi} + w \frac{\partial v}{\partial z} + \frac{uv}{r} = -\frac{1}{\rho r} \frac{\partial P}{\partial \varphi} + \nu \left[\nabla^2 v - \frac{v}{r^2} + \frac{2}{r^2} \frac{\partial u}{\partial \varphi} \right],$$~~

~~$$\frac{\partial w}{\partial t} + u \frac{\partial w}{\partial r} + \frac{v}{r} \frac{\partial w}{\partial \varphi} + w \frac{\partial w}{\partial z} = -\frac{1}{\rho} \frac{\partial P}{\partial z} + \nu \nabla^2 w,$$~~

~~$$\text{div } \mathbf{u} = \frac{1}{r} \frac{\partial}{\partial r} (ru) + \frac{1}{r} \frac{\partial v}{\partial \varphi} + \frac{\partial w}{\partial z} = 0.$$~~

where

$$u = v_r = 0$$

$$v_\varphi = v(r)$$

$$w = v_z = 0$$

$$\nabla p = \underline{0}$$

$$\nabla^2 \equiv \frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2}{\partial \varphi^2} + \frac{\partial^2}{\partial z^2}.$$

ϕ -mom. eqn:

(7)

$$0 = \left(\underbrace{\frac{d^2 u}{dr^2} + \frac{1}{r} \frac{du}{dr}}_{D^2 u} - \frac{u}{r^2} \right)$$

$$r^2 \frac{d^2 u}{dr^2} + r \frac{du}{dr} - u = 0$$

Euler eqn

$$u \sim r^\lambda$$