Chapter 5

The equations of linear elasticity

5.1 Summary of equations

• Strain-displacement relations:

$$e_{ij} = \frac{1}{2}(u_{i,j} + u_{j,i}) \tag{5.1}$$

• Equilibrium equations/equations of motion:

$$\tau_{ij,j} + F_i = \rho \frac{\partial^2 u_i}{\partial t^2} \tag{5.2}$$

• Constitutive equations:

$$\tau_{ij} = \lambda \delta_{ij} e_{kk} + 2\mu e_{ij} \tag{5.3}$$

5.2 Displacement formulation: The Navier-Lamé equations

• Solve for the displacements:

$$(\lambda + \mu)u_{k,ki} + \mu u_{i,kk} + F_i = \rho \frac{\partial^2 u_i}{\partial t^2}$$

$$(5.4)$$

or symbolically:

$$(\lambda + \mu)$$
 grad div $\mathbf{u} + \mu \nabla^2 \mathbf{u} + \mathbf{F} = \rho \frac{\partial^2 \mathbf{u}}{\partial t^2},$ (5.5)

which is equivalent to:

$$(\lambda + 2\mu)$$
 grad div $\mathbf{u} - \mu$ curl curl $\mathbf{u} + \mathbf{F} = \rho \frac{\partial^2 \mathbf{u}}{\partial t^2}$. (5.6)

• This is a system of three coupled linear elliptic PDEs for the three displacements $u_i(x_i)$.

5.3 Stress formulation: The static Beltrami-Mitchell equations

• For static deformations, we have

$$\frac{1-\nu}{1+\nu}\underbrace{\tau_{ii,jj}}_{\theta} + F_{i,i} = 0 \quad \text{or symbolically} \quad \frac{1-\nu}{1+\nu} \nabla^2 \theta + \text{div } \mathbf{F} = 0. \tag{5.7}$$

and the stresses fulfil the Beltrami-Mitchell equations:

$$\underbrace{\tau_{ij,kk}}_{\nabla^2 \tau_{ij}} + \underbrace{\frac{1}{1+\nu}}_{\theta_{,ij}} \underbrace{\tau_{kk,ij}}_{\theta_{,ij}} + \underbrace{\frac{\nu}{1-\nu}}_{\delta_{ij}} \delta_{ij} \underbrace{F_{k,k}}_{\text{div } \mathbf{F}} + F_{j,i} + F_{i,j} = 0. \tag{5.8}$$

(5.8) represents a system of six coupled linear elliptic PDEs for the six stress components τ_{ij}(x_j).
 When these have been determined, the strains can be recovered from (4.6) or (4.16). Then the displacements follow from (5.1). They are only determined up to arbitrary rigid body motions.

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5.4 Simplifications for F = const.:

For constant (or vanishing!) body force, the stress, strain and displacement components are biharmonic functions.

$$u_{i,jjkk} = 0 \tau_{ij,kkll} = 0 e_{ij,kkll} = 0 (5.9)$$

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or symbolically:

$$\nabla^4 \mathbf{u} = 0 \qquad \nabla^4 \tau_{ij} = 0 \qquad \nabla^4 e_{ij} = 0. \tag{5.10}$$

• The dilation and the trace of the stress tensor are harmonic functions:

$$u_{j,jkk} = d_{,kk} = 0$$
 $\tau_{jj,kk} = \theta_{,kk} = 0$ (5.11)

or symbolically:

$$\nabla^2 d = 0 \qquad \nabla^2 \theta = 0 \tag{5.12}$$

Note that in (5.4) – (5.8) F acts as an inhomogeneity in a system of linear equations. The system
can be transformed into a homogeneous system for u_h = u-u_p (with different boundary conditions)
if a particular solution u_p (which does not have to fulfil the boundary conditions) can be found.

5.5 Boundary conditions:

• Displacement (Dirichlet) boundary conditions: Prescribed displacement field $u_i^{(0)}$.

$$u_i|_{\partial D} = u_i^{(0)} \tag{5.13}$$

• Stress (Neumann) boundary conditions: Prescribed (applied) traction $t_i^{(0)}$ on boundary. Note that n_j is the *outer* unit normal vector on the elastic body.

$$t_i|_{\partial D} = \tau_{ij} n_j|_{\partial D} = t_i^{(0)} \tag{5.14}$$

Mixed (Robin) boundary conditions – 'elastic foundation' represented by the stiffness tensor k_{ij}
 Physically, this implies that the traction which the elastic foundation exerts on the body is proportional to the boundary displacement. This can be combined with an applied traction t_i⁽⁰⁾ as in the Neumann case.

$$(t_i + k_{ij}u_j)|_{\partial D} = (\tau_{ij}n_j + k_{ij}u_j)|_{\partial D} = t_i^{(0)}$$
(5.15)

Governing Equations in Cylindrical Polar Coordinates

• $x_1 = x = r \cos \theta$, $x_2 = y = r \sin \theta$, $x_3 = z = z$.

$$\mathbf{u} = (u_r, u_\theta, u_z), \quad e = (e_{ij}), \quad \tau = (\tau_{ij}), \quad \text{where } i, j = r, \theta, z.$$

Vector calculus:

$$\operatorname{grad} f = \frac{\partial f}{\partial r} \hat{\mathbf{r}} + \frac{1}{r} \frac{\partial f}{\partial \theta} \hat{\boldsymbol{\theta}} + \frac{\partial f}{\partial z} \hat{\mathbf{z}}, \qquad \operatorname{div} \mathbf{u} = \frac{1}{r} \frac{\partial (ru_r)}{\partial r} + \frac{1}{r} \frac{\partial u_{\theta}}{\partial \theta} + \frac{\partial u_z}{\partial z},$$
$$\operatorname{curl} \mathbf{u} = \left(\frac{1}{r} \frac{\partial u_z}{\partial \theta} - \frac{\partial u_{\theta}}{\partial z}\right) \hat{\mathbf{r}} + \left(\frac{\partial u_r}{\partial z} - \frac{\partial u_z}{\partial r}\right) \hat{\boldsymbol{\theta}} + \left(\frac{1}{r} \frac{\partial (ru_{\theta})}{\partial r} - \frac{1}{r} \frac{\partial u_r}{\partial \theta}\right) \hat{\mathbf{z}}.$$

• Stress-strain relations have the same form as in Cartesian coordinates:

$$\tau_{ij} = \lambda \delta_{ij} \operatorname{div} \mathbf{u} + 2\mu e_{ij}, \quad i, j = r, \theta, z.$$

• Stress-displacement relations:

$$\begin{split} \tau_{rr} &= \lambda \operatorname{div} \mathbf{u} + 2\mu \frac{\partial u_r}{\partial r}, \quad \tau_{\theta\theta} = \lambda \operatorname{div} \mathbf{u} + 2\mu \left(\frac{1}{r} \frac{\partial u_\theta}{\partial \theta} + \frac{u_r}{r} \right), \quad \tau_{zz} = \lambda \operatorname{div} \mathbf{u} + 2\mu \frac{\partial u_z}{\partial z}, \\ \frac{\tau_{r\theta}}{\mu} &= \frac{\tau_{\theta r}}{\mu} = \frac{\partial u_\theta}{\partial r} - \frac{u_\theta}{r} + \frac{1}{r} \frac{\partial u_r}{\partial \theta}, \quad \frac{\tau_{rz}}{\mu} = \frac{\tau_{zr}}{\mu} = \frac{\partial u_z}{\partial r} + \frac{\partial u_r}{\partial z}, \quad \frac{\tau_{\theta z}}{\mu} = \frac{\tau_{z\theta}}{\mu} = \frac{1}{r} \frac{\partial u_z}{\partial \theta} + \frac{\partial u_\theta}{\partial z} \end{split}$$

• Strain-displacement relations:

$$\begin{split} e_{rr} &= \frac{\partial u_r}{\partial r}, \qquad e_{\theta\theta} = \frac{1}{r} \frac{\partial u_{\theta}}{\partial \theta} + \frac{u_r}{r}, \qquad e_{zz} = \frac{\partial u_z}{\partial z}, \\ 2e_{r\theta} &= 2e_{\theta r} = \frac{\partial u_{\theta}}{\partial r} - \frac{u_{\theta}}{r} + \frac{1}{r} \frac{\partial u_r}{\partial \theta}, \quad 2e_{rz} = 2e_{zr} = \frac{\partial u_r}{\partial z} + \frac{\partial u_z}{\partial r}, \quad 2e_{z\theta} = 2e_{\theta z} = \frac{1}{r} \frac{\partial u_z}{\partial \theta} + \frac{\partial u_{\theta}}{\partial z}. \end{split}$$

• Equilibrium equations (statics): for the displacement formulation, use Navier's equation,

$$(\lambda + 2\mu)$$
 grad div $\mathbf{u} - \mu$ curl curl $\mathbf{u} + \mathbf{F} = \mathbf{0}$,

whereas for the stress formulation, use

$$\begin{split} \frac{\partial \tau_{rr}}{\partial r} + \frac{1}{r} \frac{\partial \tau_{r\theta}}{\partial \theta} + \frac{\partial \tau_{rz}}{\partial z} + \frac{\tau_{rr} - \tau_{\theta\theta}}{r} + F_r &= 0 \\ \frac{\partial \tau_{r\theta}}{\partial r} + \frac{1}{r} \frac{\partial \tau_{\theta\theta}}{\partial \theta} + \frac{\partial \tau_{\thetaz}}{\partial z} + \frac{2}{r} \tau_{r\theta} + F_{\theta} &= 0 \\ \frac{\partial \tau_{rz}}{\partial r} + \frac{1}{r} \frac{\partial \tau_{\thetaz}}{\partial \theta} + \frac{\partial \tau_{zz}}{\partial z} + \frac{1}{r} \tau_{rz} + F_z &= 0. \end{split}$$

• Stress boundary conditions: these are when **t** is prescribed. We have, from $t_i = \hat{n}_j \tau_{ij}$,

$$\begin{array}{lll} t_r & = & \hat{n}_r \tau_{rr} + \hat{n}_\theta \tau_{r\theta} + \hat{n}_z \tau_{rz} \\ t_\theta & = & \hat{n}_r \tau_{r\theta} + \hat{n}_\theta \tau_{\theta\theta} + \hat{n}_z \tau_{\thetaz} \\ t_z & = & \hat{n}_r \tau_{rz} + \hat{n}_\theta \tau_{\thetaz} + \hat{n}_z \tau_{zz} \end{array}$$

Governing Equations in Spherical Polar Coordinates

• $x_1 = x = r \sin \theta \cos \phi$, $x_2 = y = r \sin \theta \sin \phi$, $x_3 = z = r \cos \theta$.

$$\mathbf{u} = (u_r, u_\theta, u_\phi), \quad e = (e_{ij}), \quad \tau = (\tau_{ij}), \quad \text{where } i, j = r, \theta, \phi.$$

• Vector calculus:

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$$\operatorname{grad} f = \frac{\partial f}{\partial r} \hat{\mathbf{r}} + \frac{1}{r} \frac{\partial f}{\partial \theta} \hat{\boldsymbol{\theta}} + \frac{1}{r \sin \theta} \frac{\partial f}{\partial \phi} \hat{\boldsymbol{\phi}},$$

$$\operatorname{div} \mathbf{u} = \frac{1}{r^2 \sin \theta} \left\{ \frac{\partial}{\partial r} (r^2 \sin \theta \, u_r) + \frac{\partial}{\partial \theta} (r \sin \theta \, u_\theta) + \frac{\partial}{\partial \phi} (r u_\phi) \right\},$$

$$\operatorname{curl} \mathbf{u} = \frac{1}{r^2 \sin \theta} \begin{vmatrix} \hat{\mathbf{r}} & r \hat{\boldsymbol{\theta}} & r \sin \theta \, \hat{\boldsymbol{\phi}} \\ \frac{\partial}{\partial r} & \frac{\partial}{\partial \theta} & \frac{\partial}{\partial \phi} \\ u_r & r u_\theta & r \sin \phi \, u_\phi \end{vmatrix}.$$

• Stress-strain relations have the same form as in Cartesian coordinates:

$$\tau_{ij} = \lambda \delta_{ij} \operatorname{div} \mathbf{u} + 2\mu e_{ij}, \quad i, j = r, \theta, \phi.$$

• Stress-displacement relations:

$$\begin{split} \tau_{rr} &= \lambda \operatorname{div} \mathbf{u} + 2\mu \frac{\partial u_r}{\partial r}, \quad \tau_{\theta\theta} = \lambda \operatorname{div} \mathbf{u} + \frac{2\mu}{r} \left(\frac{\partial u_\theta}{\partial \theta} + u_r \right), \\ \tau_{\phi\phi} &= \lambda \operatorname{div} \mathbf{u} + \frac{2\mu}{r} \left(\frac{1}{\sin \theta} \frac{\partial u_\phi}{\partial \phi} + u_r + u_\theta \cot \theta \right), \quad \frac{\tau_{r\theta}}{\mu} = \frac{\tau_{\theta r}}{\mu} = \frac{\partial u_\theta}{\partial r} - \frac{u_\theta}{r} + \frac{1}{r} \frac{\partial u_r}{\partial \theta}, \\ \frac{\tau_{r\phi}}{\mu} &= \frac{\tau_{\phi r}}{\mu} = \frac{1}{r \sin \theta} \frac{\partial u_r}{\partial \phi} + \frac{\partial u_\phi}{\partial r} - \frac{u_\phi}{r}, \quad \frac{\tau_{\theta\phi}}{\mu} = \frac{\tau_{\phi\theta}}{\mu} = \frac{1}{r \sin \theta} \frac{\partial u_\theta}{\partial \phi} + \frac{1}{r} \frac{\partial u_\phi}{\partial \theta} - \frac{u_\phi \cot \theta}{r}. \end{split}$$

• Strain-displacement relations:

$$\begin{split} e_{rr} &= \frac{\partial u_r}{\partial r}, \qquad e_{\theta\theta} = \frac{1}{r} \frac{\partial u_\theta}{\partial \theta} + \frac{u_r}{r}, \qquad e_{\phi\phi} = \frac{1}{r \sin \theta} \frac{\partial u_\phi}{\partial \phi} + \frac{u_r}{r} + \frac{u_\theta \cot \theta}{r}, \\ \\ 2e_{r\theta} &= 2e_{\theta r} = \frac{\partial u_\theta}{\partial r} - \frac{u_\theta}{r} + \frac{1}{r} \frac{\partial u_r}{\partial \theta}, \quad 2e_{r\phi} = 2e_{\phi r} = \frac{1}{r \sin \theta} \frac{\partial u_r}{\partial \phi} + \frac{\partial u_\phi}{\partial r} - \frac{u_\phi}{r}, \\ \\ 2e_{\phi\theta} &= 2e_{\theta\phi} = \frac{1}{r \sin \theta} \frac{\partial u_\theta}{\partial \phi} + \frac{1}{r} \frac{\partial u_\phi}{\partial \theta} - \frac{u_\phi \cot \theta}{r}. \end{split}$$

• Equilibrium equations (statics): for the displacement formulation, use Navier's equation,

$$(\lambda + 2\mu)$$
 grad div $\mathbf{u} - \mu$ curl curl $\mathbf{u} + \mathbf{F} = \mathbf{0}$,

whereas for the stress formulation, use

$$\begin{split} \frac{\partial \tau_{rr}}{\partial r} + \frac{1}{r} \frac{\partial \tau_{r\theta}}{\partial \theta} + \frac{1}{r \sin \theta} \frac{\partial \tau_{r\phi}}{\partial \phi} + \frac{2\tau_{rr} - \tau_{\theta\theta} - \tau_{\phi\phi} + \cot \theta \, \tau_{r\theta}}{r} + F_r &= 0 \\ \frac{\partial \tau_{r\theta}}{\partial r} + \frac{1}{r} \frac{\partial \tau_{\theta\theta}}{\partial \theta} + \frac{1}{r \sin \theta} \frac{\partial \tau_{\theta\phi}}{\partial \phi} + \frac{3\tau_{r\theta} + (\tau_{\theta\theta} - \tau_{\phi\phi}) \cot \theta}{r} + F_{\theta} &= 0 \\ \frac{\partial \tau_{r\phi}}{\partial r} + \frac{1}{r} \frac{\partial \tau_{\theta\phi}}{\partial \theta} + \frac{1}{r \sin \theta} \frac{\partial \tau_{\phi\phi}}{\partial \phi} + \frac{3\tau_{r\phi} + 2\tau_{\theta\phi} \cot \theta}{r} + F_{\phi} &= 0. \end{split}$$

• Stress boundary conditions: these are when **t** is prescribed. We have, from $t_i = \hat{n}_i \tau_{ii}$,

$$\begin{array}{lll} t_r & = & \hat{n}_r \tau_{rr} + \hat{n}_\theta \tau_{r\theta} + \hat{n}_\phi \tau_{r\phi} \\ t_\theta & = & \hat{n}_r \tau_{r\theta} + \hat{n}_\theta \tau_{\theta\theta} + \hat{n}_\phi \tau_{\theta\phi} \\ t_\phi & = & \hat{n}_r \tau_{r\phi} + \hat{n}_\theta \tau_{\theta\phi} + \hat{n}_\phi \tau_{\phi\phi} \end{array}$$