# Multi-domain active sound control and noise shielding

H. Lim<sup>a)</sup>

Acoustics Research Centre, University of Salford, Salford, Greater Manchester, M5 4WT, United Kingdom

# S. V. Utyuzhnikov

School of Mechanical, Aerospace and Civil Engineering, University of Manchester, M13 9PL, United Kingdom

# Y. W. Lam

Acoustics Research Centre, University of Salford, Salford, Greater Manchester, M5 4WT, United Kingdom

#### A. Turan

School of Mechanical, Aerospace and Civil Engineering, University of Manchester, M13 9PL, United Kingdom

(Received 16 April 2010; revised 15 November 2010; accepted 2 December 2010)

This paper describes an active sound control methodology based on difference potentials. The main feature of this methodology is its ability to automatically preserve "wanted" sound within a domain while canceling "unwanted" noise from outside the domain. This method of preservation of the wanted sounds by active shielding control is demonstrated with various broadband and realistic sound sources such as human voice and music in multiple domains in a one-dimensional enclosure. Unlike many other conventional active control methods, the proposed approach does not require the explicit characterization of the wanted sound to be preserved. The controls are designed based on the measurements of the total field on the boundaries of the shielded domain only, which is allowed to be multiply connected. The method is tested in a variety of experimental cases. The typical attenuation of the unwanted noise is found to be about 20 dB over a large area of the shielded domain and the original wanted sound field is preserved with errors of around 1 dB and below through a broad frequency range up to 1 kHz. © *2011 Acoustical Society of America*. [DOI: 10.1121/1.3531933]

PACS number(s): 43.50.Ki, 43.40.Sk, 43.55.Dt, 43.55.Br [KVH]

Pages: 1–9

## I. INTRODUCTION

Active control of sound is a technique for altering acoustic fields to a desired one by introducing controllable active secondary sound sources called controls. An example problem formulation in this area involves a given region of space (bounded or unbounded) to be shielded from unwanted external noise by the active controls. The controls establish an active boundary, shielding the region from the noise. This specific strategy for noise cancellation by means of active boundary controls is called active shielding (AS). The overall problem of active noise cancellation becomes more complicated if, along with the unwanted noise, a wanted sound component is present inside the protected region.

Generally, existing conventional active control methods, for example those developed by Nelson and Elliott,<sup>1</sup> and Kincaid *et al.*,<sup>2,3</sup> require an accurate description of the original noise source in order to devise a global cancellation solution. When the measurement is carried out in close proximity of the shielded domain and the noise source is not measured, significant noise reduction can generally be achieved only locally in conventional approaches.<sup>1</sup> Often in practice it is not feasible to measure the physical values of the original noise source since the noise source is not always accessible.<sup>4</sup> In addition, the transfer function of the sound through the problem domain has to be taken into account to achieve a global noise cancellation solution. This is particularly difficult if the medium of propagation is inhomogeneous. To overcome these limitations and associated practical difficulties, the difference potential method (DPM) proposed here can provide a convenient solution. The theoretical concept is based on the method described in Refs. 5 and 6. It allows one to obtain a general solution to the AS problem for arbitrary geometries, properties of the medium, or boundary conditions.

Theoretically, the DPM allows us to reduce a boundary value problem set in a complex domain to a boundary equa-tion. Its key characteristics include the capability to cancel out the unwanted noise in a large region of the shielded do-main, while requiring no detailed knowledge of either the sound transfer function for the problem domain or the noise sources. The only input data needed by the methodology are acoustic quantities at the perimeter of the protected region (in practice they can be measured). By requiring only this limited input data, the unique characteristics of the method can provide a practical and cost-effective control system. Moreover, these quantities may pertain to an overall acoustic field composed both of unwanted and wanted components. The methodology automatically differentiates between the two. The method suggested by Jessel and Mangiante,<sup>7,8</sup> and Canevet,<sup>9</sup> hereby called the JMC method,<sup>10</sup> yields solutions for global noise cancellation in a similar way when only the unwanted noise is present in the protected domain. The main difference between the approaches based on the DPM and JMC is that only the former provides the advantages of pres-ervation of the wanted sound and volumetric noise cancella-tion through an entire shielded domain when the total field 

0001-4966/2011/129(2)/1/9/\$30.00

<sup>&</sup>lt;sup>a)</sup>Author to whom correspondence should be addressed. Electronic mail: h.lim@edu.salford.ac.uk

2

composed of both the wanted sound and noise is measured on the boundary. In addition, the DPM-based approach allows a shielded domain to be multiply connected. This capability is potentially very useful for applications related to noise control and room acoustics, as it enables protection of the predefined space against the noise coming from the outside, while at the same time not interfering with the ability of the listener to listen to wanted sound or communicate inside the room. Although this technique has been introduced and studied theoretically in previous publications,<sup>11,12</sup> the unique feature that allows us to preserve the wanted sounds in multi-domain has never been experimentally studied and published in literature. The main focus of the paper is an implementation of the novel AS technique based on the difference potentials to multi-domain tests with broadband signals. In practice a sound field is generally composed of broadband frequencies rather than a pure tone. A broadband sound source may cause extreme fluctuations of sound pressure at some frequencies due to resonances and anti-resonances in a duct closed by rigid terminations. Quite often a control system based on many other conventional AS methods fails to achieve efficient cancellation of noise at resonances. The reason is that the sensitivity of their solutions to errors is too high at resonances. With that in mind, the characteristics and practical limitations of the approach over a broad frequency band will be evaluated and discussed in this paper. As we are in an early stage of the experimental investigation for the method, a real-time control system has not been implemented yet. The overall system is assumed to be linear time-invariant and repeatable in the experiment.

## **II. THEORETICAL FORMULATION**

Details of the theoretical formulation have already been described in previous publications.<sup>13,14</sup> Only a brief outline of the concept is given here to help the understanding relevant to the experimental design. Assume that the propagation of sound is governed by a linear partial differential equation or system in a domain  $D_{o}$ . The sound field is composed of both adverse noise and wanted sound. We formulate the AS problem as follows. It is required to find such additional sources that the solution to the modified problem coincides with the wanted sound in a subdomain D, i.e. D is part of  $D_{o}$ that is to be shielded. It is important to note here that the domain D is not necessarily simply connected. It is also noted that the reverberation field of wanted sound is also considered as part of the wanted sound. From a practical point of view, the additional sources, which are the "controls," cannot be immediately realized because the solution assumes the distribution of controls to be continuous. Here, a discrete distribution of the controls can be obtained via the theory of difference potentials (see Refs. 15 and 16 for details). It can also be interpreted as a discrete approximation of the continuous solution. For the AS solution it is sufficient to have access only to the trace of the total acoustic field on the boundary of the domain D. In particular, no knowledge of the actual sources (wanted and unwanted) is required. Thus, such active controls are more practical then the trivial solution of having a control equals to the ideal negative of the unwanted source, which is difficult to implement even if the adverse sources are explicitly available.

181

182

185

186

187

188

189

207

208

209

210

211

212

213

214

215

216

217

218

219

220

221

222

223

224

225

226

227

228

229

230

231

232

235

236

239

240

183 It is important to emphasize that the control sources are 184 obtained for the general case and do not require knowledge of the Green's function of the problem. It has been shown that the space AS solution is based on the knowledge of the total sound pressure and the normal component of the particle velocity on boundary surface of the shielded domain.<sup>15,16</sup> The general solution is applicable in the general case of 3D flow field.

190 The solution can be illustrated for a one-dimensional 191 case, in which the primary noise sources are situated in the 192 area  $-\ell < x < \ell$ , whereas the secondary (control) sources G 193 are to be placed at  $x = -\ell$  and  $x = \ell$  to protect the domains 194  $x < -\ell$  and  $x > \ell$ . These domains are interpreted as subdo-195 mains of a single multiply connected domain D. We suppose 196 the field is monochromatic and, in the frequency domain, the 197 generated wanted field is represented by sound pressure 198  $A_1 e^{-jkx}$  if  $x < -\ell$  and  $A_2 e^{jkx}$  if  $x > \ell$ . In turn, it is assumed 199 that the noise is generated in such a way that field  $B_1 e^{ikx}$ 200 propagates toward domain  $x < -\ell$ , while field  $B_2 e^{-jkx}$ , to-201 ward domain  $x > \ell$ . For simplicity, anechoic terminations 202 are assumed in the example so that there are no reflections 203 from the ends. It should be noted that the general solution 204 method itself can be applied to cases with any arbitrary 205 terminations. 206

Let us put both a monopole and dipole at each of the two points  $x = -\ell$  and  $x = \ell$ . Assume that the amplitudes of the monopoles and dipoles are given by  $q_i$  and  $b_i$ , respectively, where i = 1 corresponds to  $x = -\ell$ , while i = 2 in the case of  $x = \ell$ . In addition, suppose both the dipoles are oriented toward x = 0.

The field generated by all the primary and secondary sources is given by

$$\bar{p}(x) = \mathsf{A}_1 e^{-jkx} + \mathsf{A}_2 e^{jkx} + \mathsf{B}_1 e^{jkx} + (q_1 - b_1) e^{jk(x+\ell)} + (q_2 + b_2) e^{jk(x-\ell)},$$

if  $x < -\ell$  and

$$\bar{p}(x) = \mathsf{A}_1 e^{-jkx} + \mathsf{A}_2 e^{jkx} + \mathsf{B}_2 e^{-jkx} + (q_1 + b_1) e^{-jk(x+\ell)} + (q_2 - b_2) e^{-jk(x-\ell)},$$

if  $x > \ell$ .

We require only the field  $\bar{p}(x) = A_1 e^{-jkx} + A_2 e^{jkx}$  to be composed of the two wanted sounds in the protected domains  $x < -\ell$  and  $x > \ell$ . It should be noted that this requirement is fundamentally different from that of an active absorber, which would have required only  $A_1 e^{-jkx}$  in the domain  $x < -\ell$  and  $A_2 e^{jkx}$  in the domain  $x > \ell$ .

Then, we arrive at the following requirements,

$$(q_1 - b_1)e^{jk\ell} + (q_2 + b_2)e^{-jk\ell} + B_1 = 0,$$
 (1)  $\begin{array}{c} 233\\234 \end{array}$ 

if  $x < -\ell$  and

$$(q_1 + b_1)e^{-jk\ell} + (q_2 - b_2)e^{jk\ell} + B_2 = 0,$$
 (2)  $\frac{237}{238}$ 

if  $x > \ell$ .

Lim et al.: Multi-domain active sound control

J. Acoust. Soc. Am., Vol. 129, No. 2, February 2011

Stage:

301

304 305 Here,  $\vec{n}$  is the external normal to the boundary  $\Gamma$  of the protected 306 domain and  $\delta(\Gamma)$  is the delta-function assigned to the surface  $\Gamma$ . 307 In the example, the coordinate of the normal to the boundary  $\Gamma$ 308 at  $x = -\ell$  equals 1, while at  $x = \ell$  it is equal to -1. 309

In practical applications, the point sources (5), should be approximated by spatially extended terms.<sup>15</sup> For example, the controls at point  $x = \ell$  is represented by

$$q_{\text{vol},\ell}^{(h)} = -\frac{\Theta_h(x-\ell)}{h}u_\ell,$$
313
314

$$b_{\text{vol},\ell}^{(h)} = -\frac{\Theta_h(x-\ell)}{h} p_\ell.$$
(7)

Here,  $\Theta_h(x) \equiv \theta(h/2 - x)\theta(h/2 + x)$ , where h denotes the finite difference step and  $\theta(x)$  is the indicator function; the particle velocity  $u_{\ell}$  and sound pressure  $p_{\ell}$ , respectively, should be measured near the point  $x = \ell$ . Accordingly, these controls are represented by the volume velocity per unit volume and the force per unit volume, see Ref. 1. In an experimental setting, h in Eq. (7) corresponds to the thickness of the source, <sup>15,20</sup> the control  $q_{vol}^{(h)}$  is implemented as an acoustic monopole and the control  $b_{vol}^{(h)}$  is implemented as a dipole. The thickness of the source should be adequately smaller than the wavelength. This follows from the theoretical approximation  $kh \ll 1.^{15}$ 

330 In Ref. 15, for a simply connected domain it is shown 331 that the controls (6) preserve the reverberation field of the 332 wanted sound. The mechanism behind this is as follows. The 333 controls attenuate any field coming into the domain to be 334 shielded and, at the same time, generate only the field inside 335 the domain that is exactly required to restore the reverbera-336 tion of the wanted sound there. Thus, the AS controls are 337 "transparent" to the reflected component of the wanted sound 338 coming into the protected domain. The same conclusion is 339 applicable to a multiply connected domain. For instance, in 340 the example in question, assume that there is a rigid termina-341 tion on the right-hand side at  $x = \ell_1 > \ell$ . Then, we can inter-342 pret any reverberation of wanted sound generated in the 343 domain  $\ell < x < \ell_1$  as just the wanted sound from this do-344 main. In addition, there is no reverberation of noise because 345 of its attenuation in the domain  $\ell < x < \ell_1$ . Then, the prob-346 lem can be reduced to the example considered above except 347 the reverberation of the wanted sound generated in the do-348 main  $x < -\ell$ . In contrast to the previous case, the input 349 (measured) data at  $x = -\ell$  change due to the contribution of 350 the reflected field of the wanted sound from the left-hand 351 side. However, as noted above, the appropriate additional 352 secondary sources do not damage the reverberation field in 353 domain  $x < -\ell$ . In turn, the controls at  $x = \ell$  are transparent 354 to the wanted field coming from the left-hand side and its 355 reverberation holds inside the protected domain  $\ell < x < \ell_1$ . 356

In the experimental implementation of the AS solution, 357 there are also some restrictions depending on the frequency 358 of acoustic signals generated. In practice, to maximize the ef-359 ficiency in attenuation in 3D space, the optimum distribution 360

The particle velocity and sound pressure before the control are given by

$$\begin{split} \bar{p}(-\ell) &= \mathsf{A}_{1}e^{jk\ell} + \mathsf{A}_{2}e^{-jk\ell} + \mathsf{B}_{1}e^{-jk\ell},\\ \bar{u}(-\ell) &= \frac{1}{\rho c}(\mathsf{A}_{1}e^{jk\ell} - \mathsf{A}_{2}e^{-jk\ell} - \mathsf{B}_{1}e^{-jk\ell}),\\ \bar{p}(\ell) &= \mathsf{A}_{1}e^{-jk\ell} + \mathsf{A}_{2}e^{jk\ell} + \mathsf{B}_{2}e^{-jk\ell},\\ \bar{u}(\ell) &= \frac{1}{\rho c}(\mathsf{A}_{1}e^{-jk\ell} - \mathsf{A}_{2}e^{jk\ell} + \mathsf{B}_{2}e^{-jk\ell}). \end{split}$$
(3)

Next, having formally applied the secondary sources derived in Ref. 15 for a simply connected domain, at each of the two boundary points  $(x = \pm \ell)$ , we arrive at the following controls,

$$q_{1} = \frac{\rho c \bar{u}(-\ell)}{2}, \qquad q_{2} = -\frac{\rho c \bar{u}(\ell)}{2}, \\ b_{1} = \frac{\bar{p}(-\ell)}{2}, \qquad b_{2} = \frac{\bar{p}(\ell)}{2}.$$
(4)

Alternatively, the controls (4) can simply be postulated. Then, substituting Eqs. (3) into (4), one can verify that conditions (1) and (2) are valid.

To analyze the operation of the controls, let us consider, for example, the domain  $x < -\ell$ . From Eq. (1) one can obtain that the control sources at point  $x = -\ell$  generate the following field,

$$\bar{p}(x) = -\{(q_2 + b_2)e^{jk(x-\ell)} + \mathsf{B}_1e^{jkx} + \mathsf{A}_2e^{jkx}\} + \mathsf{A}_2e^{jkx}\}$$

The above equation is written to show explicitly that the controls at  $x = -\ell$  attenuate any field coming into domain  $x < -\ell$ , on the one hand, and restore the wanted field (the last term with  $A_2$ ) from the right-hand side, on the other hand.

Overall, the governing acoustics equations after the control are written as

$$\frac{\partial p}{\partial t} + \rho c^2 \frac{\partial u}{\partial x} = \rho c^2 q_{\text{vol}} + f_p$$
$$\frac{\partial u}{\partial t} + \frac{1}{\rho} \frac{\partial p}{\partial x} = \frac{b_{\text{vol}}}{\rho} + f_u,$$

where the controls are given by

$$q_{\text{vol}}(x) = u(-\ell)\delta(x+\ell) - u(\ell)\delta(x-\ell),$$
  

$$b_{\text{vol}}(x) = p(-\ell)\delta(x+\ell) - p(\ell)\delta(x-\ell).$$
(5)

Here,  $f_p$  and  $f_u$  are the appropriate source functions,  $\delta(x)$  is the 1D delta-function, determined in the space of distributions (see, e.g., Ref. 17). The controls  $q_{vol}$  and  $b_{vol}$  of Eq. (5) depend on the particle velocity and sound pressure, respectively. It is to be noted that, as soon as we introduce the delta-function, we should consider the solution to the problem in the generalized sense.<sup>18</sup> Alternatively, the delta-function should be approximated by its counterpart in the space of standard functions.

One can see, the controls (5) are a partial case of the general solution<sup>19</sup> in 3D case,

J. Acoust. Soc. Am., Vol. 129, No. 2, February 2011

241

302

310

311

312

315

316

317

318

319

320

321

322

323

324

325

326

327

328

329

of the control sources on boundary surfaces has to be defined. 362 Optimization of the control sources with respect to different 363 criteria has been studied by Loncaric and Tsynkov in Ref. 21. 364 It is to be noted that the described approach has recently been 365 extended to a nonstationary formulation and arbitrary degree 366 of sound control in Refs. 19 and 22, respectively. 367

# **III. EXPERIMENTAL RESULTS**

361

368

369

370

371

406

407

408

409

410

418

419

420

# A. AS of a simply connected domain

Regarding the validation of the AS solution based on 372 difference potentials, it is helpful to start by analyzing a one-373 374 dimensional linear problem in the first instance. The solution 375 for the AS problems either with or without the wanted sounds have previously been experimentally validated with 376 pure tone sound sources in a duct, and the results were 377 378 reported in Ref. 20. Following this work, the experiment is now extended to cover broadband sound fields, including 379 380 multiple resonance regions. This is done in a cylindrical duct manufactured of polypropylene tubing, which is sufficiently 381 rigid to allow losses through the duct walls to be neglected.<sup>23</sup> 382 As most AS techniques are more effective in general at 383 lower frequency rather than at higher frequency,<sup>1,24</sup> the fre-384 385 quency range is limited to below 1 kHz in the experiment. 386 The duct is 4.42 m in length. Its inner diameter is 0.17 m, which allows it to be approximated acoustically as one-387 F1 388 dimensional up to a frequency of about 1 kHz. Figure 1(a) 389 illustrates the excitation of an unwanted noise source at the 390 right-end of the duct, and wanted sound source inside of the 391 shielded domain in a one-dimensional, rigid walled cylindri-392 cal enclosure. The domain to be shielded occupies approxi-393 mately one third of the entire volume of the enclosure on the left-hand side. Any sound field having its acoustic source sit-394 395 uated in the shielded domain is defined as "wanted." Sources outside are otherwise "unwanted" or "noise." The sound 396 field includes reflections of the sound in the tube. In other 397 398 words, the wanted field is the combination of the sound 399 directly emitted from the wanted sound source and its reverberation within the tube. The fields of the wanted sound and 400 401 noise, and the outputs of the controls satisfy the plane wave conditions. Measuring sensors and a control source unit are 402 located at the boundary of the shielded domain, at x = 0. The 403 404 sound pressure is measured along the axis of the duct in all the experiments. 405

> In the measuring process, before obtaining the AS solutions for a given problem, directional and non-directional components of the sound field are measured using a Brüel & Kjær PULSE Sound & Vibration analyzer with the control sources off. The former is the normal component of the par-



FIG. 1. Experimental setup for the test: (a) noise cancellation and preservation of wanted sound and (b) direction of an acoustic dipole.

ticle velocity  $u_o$ , and the latter is the acoustic pressure  $p_o$  of the total field at the boundary. Then, the directional component measured defines a non-directional AS control source which is a monopole. On the other hand, the non-directional component measured is used to define a dipole control source which is directional.

421

422

423

424

425

426

427 The control sound field is derived from the measure-428 ments of the total field of the unwanted noise and the wanted 429 sound on the boundary of the shielded domain. Unlike other 430 approaches, for the preservation of a desirable sound and 431 cancellation of noise, the procedure does not require any 432 additional explicit information regarding the wanted sound 433 or the system. In contrast, previous studies, e.g. Refs. 25 and 434 26, for similar control cases required either the wanted sound 435 or the unwanted noise to be absent in the measurement. In a 436 recent paper, directional measuring devices, i.e. directional 437 microphones, have been used in order to identify the wanted 438 component apart from the total field.<sup>27</sup> This is not required 439 in our case because the wanted and unwanted components 440 are discriminated automatically even in the case when the 441 reverberation of the wanted sound propagates from the same 442 direction of the unwanted one<sup>15</sup> (see also example given in 443 the Theoretical Formulation section). The measurement of 444 the particle velocity at the boundary and the difference 445 potential formulation are able to capture the difference in the 446 location of the wanted and unwanted sources automatically. 447 When the control sources are mounted on the boundary, the 448 direction of the dipole source defines the inside and outside 449 of a shielded domain. For this reason the direction of the 450 dipole source must be perpendicular to the boundary and 451 pointed out from the shielded domain [Fig. 1(b)]. The sound 452 generation system consists of loudspeakers, amplifiers, and a 453 PC with a multi-channel sound card. The measured values, 454 adjusted for the transfer function of the signal generator, are 455 used to calculate offline the control source signals based on 456 the difference potential theory. The source strengths of the 457 controls b and q related to the reference signal  $V_{ref}$  are 458

$$\hat{b} = \frac{\hat{p}_o A_s}{H_d}, \quad \hat{q} = \frac{(\hat{u}_o \vec{n}) A_s}{H_m}.$$
(8)
459
460
460

462 Here  $A_s$  is a cross-sectional surface area,  $H_d$  is the transfer 463 function of the dipole source signal generator,  $H_m$  is the 464 transfer function of the monopole source signal generator, 465 and  $\vec{n}$  is a unit normal vector on the boundary surface. In (8) 466  $\hat{b} = b/V_{\text{ref}}, \hat{q} = q/V_{\text{ref}}, \hat{p}_o = p_o/V_{\text{ref}}, \text{ and } \hat{u}_o = u_o/V_{\text{ref}}.$ 467

Then, the control source signals (6) are then saved as 468 phase-synchronous .wav files which can be played back 469 using a multi-channel compatible wave editor. This calcula-470 tion is not carried out in real-time in the current setup. The 471 resulting attenuation of the unwanted sound and the preser-472 vation of the wanted sound are studied by measuring the 473 total field composed with the wanted, unwanted, and control 474 sound all together in the duct. 475

#### B. Sensitivity analysis

478 In the first instance, a quick demonstration of the ability 479 of the AS method to attenuate broadband unwanted noise is 480

Lim et al.: Multi-domain active sound control

476

477



FIG. 2. Attenuation of the unwanted sound in the shielded domain. Left-hand graph is a zoom-in of the low frequency results.

496 shown in Fig. 2. In this initial experiment, wanted sound is 497 not included, and a broadband linear unidirectional swept 498 sine signal is used to generate a noise field. The signal is 499 swept forward in frequency up to 1 kHz with a rate of 500 380 Hz/s. The signal is generated through the signal genera-501 tor with a sampling rate of 2560 Hz and resolution of 16 502 bits/sample. When the AS control system is applied to a duct 503 where only an unwanted sound field propagates, it is found 504 that the solution is able to achieve an attenuation of about 505 12-18 dB in the shielded domain over a large frequency 506 range between 50 and 1 kHz except at some points, mostly 507 at anti-resonances. This is due to the fact that the sound level 508 at anti-resonances is very low. Hence the error in the mea-509 surement is relatively high. The low attenuation between 510 580 and 700 Hz is another example of very small sound pres-511 sures at the boundary of the shielded domain at those fre-512 quencies when the boundary is close to the nodal plane of 513 the predominant resonant mode. Overall, the result shows 514 that the general solution which was used for the experiments 515 with pure tone sources<sup>20</sup> is also effective with broadband 516 noise, as long as the sound field is strong enough to be meas-517 ured accurately at the boundary of the shielded domain. 518

481

482

483 484

485

486

487 488

489

490

491

492

493

494

495

F2

The sensitivity analysis (SA) has two objectives, i.e. to 519 estimate the sensitivity to changes in the input parameters, 520 and to identify the dominant sources of error (uncertainty) 521 affecting the resulting attenuation. The investigation evalu-522 ates the quality of the control system, such as functionality 523 and reliability in operation against uncertainties. The analy-524 sis is essential for the development of guidelines for the 525 practical use of the method. The error sources in the AS 526 method can be classified largely in two groups, one related 527 to position and the other to time. For instance, errors in the 528 measuring position,  $\Delta x$ , and the separation between loud-529 speakers forming a dipole,  $\Delta d$ , may exist in the realization of 530 the controls. It is not uncommon that real environment does 531 not allow for physical devices to be mounted at exactly the 532 desired positions. These kinds of errors are concerned with 533 the spatial aspect of the system. On the other hand, errors in 534 time delay,  $\Delta t$ , and phase error,  $\Delta \phi$ , can occur in digital sig-535 nal processing or measuring equipment. In addition, changes 536 of the input and output system responses with time can cause 537 errors too, if the system is not controlled adaptively in real-538 time. The initial system parameters may also be changed by 539 the introduction of the control sources. The purpose of the 540

SA is to prove the robustness of the control system due to systematic small errors. However, this analysis does not look into time varying errors.

559 Total deviation of the overall attenuation  $\delta \eta$  can be esti-560 mated as follows:  $\delta \eta = \sum |T_i| \cdot \delta n_i$ , where the problem is 561 assumed to be linear. Figure 3 shows the sensitivity  $|T_i|$  with 562 respect to the estimated changes in different variables  $n_i$ . 563 The variables  $n_i$  represent factors, namely  $\Delta x$ ,  $\Delta t$ ,  $\Delta d$ , and 564  $\Delta \varphi$ , that are perceived to have a strong influence on the ac-565 curacy of the experiment.  $\Delta x$  is related to the size of the 566 measuring microphone used in the experiment, in this case a 567 quarter-inch microphone.  $\Delta x$  is assumed to be half the diam-568 eter of the microphone head, i.e. 0.3175 cm.  $\Delta t$  is the mini-569 mum controllable time step which generally depends on the 570 limited time resolution of the digital signal generator and the 571 sampling frequency used in the measurement. For this calcu-572 lation, the sampling frequency is assumed to be 2048 Hz, 573 which gives  $\Delta t$  as 0.488 ms.  $\Delta d$  is the assumed error due to 574 uncertainty in the actual distance between the centers of the 575 loudspeakers composing the dipole source and is taken as 576 1cm in this analysis.  $\Delta d$  includes the uncertainty of the 577



FIG. 3. Deviations of attenuation according to estimated changes in  $\Delta x$ ,  $\Delta t$ , 599  $\Delta d$ , and  $\Delta \varphi$ .

541

542

543

544

545

546

547

548

549

550

551

552

553

554

555 556

557

558

F3

effective acoustic centers of the drivers composing the dipole source. The uncertainty in the overall phase  $\Delta \varphi$ , which is assumed to be a frequency independent error, up to 1 kHz is assumed to be 1° for the purpose of the analysis.

601

602

603

604

605

606

607

608

609

610

611

612

613

614

615

616

617

618

619

620

621

622

623

624

625

626

627

659

660

6

The result of the SA shows that the dominant parameters affecting the change of attenuation, when realistic values of uncertainty are used, are  $\Delta t$  and  $\Delta \varphi$ , as can be seen in Fig. 3.  $\Delta \varphi$  is usually manifested also as a time delay in the time domain. Therefore, the time signatures describing the control sources have to be dealt with in high accuracy. A typical modern digital signal processing system is able to sample input data at frequency higher than 2048 Hz, which corresponds to a  $\Delta t$  of 0.488 ms. The size of the measuring device used for measurement causes acceptable error in attenuation, as long as the sound field is measured as close as possible to the effective center of the control source unit. As stated above, the time delay error is the most dominant variable as compared to the other error parameters. These results are consistent with the theoretical SA performed in Ref. 28.

In further experiments described in later sections, broadband wanted sound fields are included to test the efficiency of the AS method in different configurations requiring preservation of wanted sound.

# C. Active control of multiple sound fields in multi-domain

F4 628 Figure 4 describes the experiment which demonstrates 629 the possibility of using AS to preserve multiple wanted 630 sounds in multi-domains. In the experiment, two shielded 631 domains are defined in both left and right ends of the duct. 632 Two different sources of the wanted sound are generated, 633 one inside each shielded domain. In addition, a noise source 634 is activated in the space between the two shielded domains. 635 To test the capability of the method in more practical cases, 636 human voice and a music track are used in the left- and 637 right-hand shielded domain, respectively, to generate the 638 broadband wanted sounds in the experiment. The wanted 639 sound signals are captured from recorded audio tracks. To fit 640 into the test frequency range of the duct, frequency contents



FIG. 4. Experimental setup for the test with two wanted sounds and one noise source in a multi-domain setting.

of the wanted sound sources that are higher than 1 kHz are attenuated by a low-pass filter. The wanted sounds are chosen to have practically different acoustic characteristics than the unwanted sound, which is generated by a white noise signal, in the experiment. The different time signatures of the output sound fields are illustrated further in this section.

To make the experimental model more general, the ge-668 ometry of the system is designed to be asymmetric with 669 respect to the size of the domains and location of the noise 670 source. AS controls are mounted on the boundaries between 671 the unwanted noise field and the shielded domains. To gener-672 alize the experiment further, and to take advantage of the 673 potential-based method's ability to work without precise 674 knowledge of system conditions, terminations with unknown 675 finite impedance at both ends are used. In the experiment, 676 the unknown impedance condition is implemented by put-677 ting approximately four inches thick generic fibrous sound 678 absorbing material on the rigid plate at each end of the tube. 679 The properties of the fibrous material are not known and are 680 not needed in the potential-based approach.

681 To apply Eqs. (1) and (2), the study considers the 682 shielded domains 1 and 2 together as one multiply connected 683 domain D. Thus, the boundary of the domain D contains two 684 parts. The experiment convincingly demonstrates that the 685 potential-based AS automatically extracts all the necessary 686 information about the system and the unwanted noise itself 687 from the measurements performed at each boundary. The 688 source strengths of the control monopoles and dipoles are 689 obtained by substituting the measured quantities, particle ve-690 locity  $u_o$ , and pressure  $p_o$ , of the total sound field at each 691 boundary into the equations for the strengths of the monop-692 oles and dipoles. 693

Figure 5 illustrates the change of the total sound pres-694 sure level in one of the shielded domains, at location  $(\times)$ , 695 specified in Fig. 4, before and after the control sources are 696 activated on both boundaries, and shows the similarity 697 between the result with AS activated and the original wanted 698 sound in the frequency domain. The symbols  $\blacklozenge$  in Fig. 5 699 show the initial sound pressure distribution when the noise 700 and wanted sounds are both on, while the control sources are 701 still off. The symbols + represent the distribution of the net sound pressure when the noise is canceled out by the con-703 trols. The net sound pressure + can be compared with the 704 wanted sounds, shown by the symbols  $\bigcirc$  in Fig. 5. When 705 the control sources are activated, the control system attains 706 an overall attenuation of around 21 dB in both the left- and right-hand shielded domains. Moreover, the net sound pres-708 sure after the potential-based controls switched on generally 709 resembles closely the original wanted sound pressure at each 710 of the measurement positions in the shielded domains. The 711 next figure shows a clearer picture with the same results pre-712 sented in 1/3-octave bands. 713

Figure 6(a) shows the similarity between the original wanted sound pressure  $\bigcirc$ , and the result,  $\Delta$  when the controls are switched on. In the experiment, a challenging condition is set up by introducing a significantly bigger unwanted sound pressure than the wanted one (about 10 dB higher), so that the results can give a reliable guidance of the attenuation that can be achieved in practice when the wanted sound has 714 715 716 717 718 718 719 720 F5

661





FIG. 5. Preservation of the wanted sound in one of the shielded domains;  $\bigcirc$ : wanted sound pressure,  $\blacklozenge$ : the sound pressure of noise and wanted sound without control, and +: total sound pressure of noise and wanted sound, and controls. Bottom graph is a zoom-in of the result in the 320–500 Hz frequency range.

been seriously contaminated by strong unwanted noise. In spite of significant level differences between the unwanted noise and the wanted sound pressure, which are denoted by \*and  $\bigcirc$ , respectively, in each frequency band in Fig. 6(a), the wanted sound pressure is mostly preserved after the controls

781 are switched on. Figure 6(b) shows the error between the 782 result and the original wanted sound in decibel scale against 783 the level difference between the unwanted noise and wanted 784 sound pressure. Obviously, when the unwanted noise 785 becomes stronger relative to the wanted sound, the error 786 increases slightly due to the decrease in signal to noise ratio. 787 However, even when the noise is up to 15 dB stronger, the 788 errors is still less than about 1 dB. When the difference is 789 higher than 15 dB, i.e. when the signal to noise ratio is below 790 -15 dB, the error increases rapidly due to inherent errors in 791 the measurement system. 792

To further support and enhance the experimental evidence of the preservation of the wanted sounds, the results are also studied in the time domain. Figure 7 illustrates the time signatures of data described above in Figs. 5 and 6.

793

794

795

F7

796 The solid line in Fig. 7(a) shows the initial sound pres-797 sure when the noise and wanted sounds are both on, while 798 the control sources are still off. Figure 7(b) represents the 799 net sound pressure field when the noise is canceled out after 800 the activation of the AS control sources. For comparison, the 801 original wanted sound is separately measured at the same 802 reference position when both the AS control and the 803 unwanted noise sources have been turned off. This is shown 804 in Fig. 7(c). The shielded result and the original wanted 805 sound are overlaid in Fig. 7(d) for the time period between 806 2.2 and 2.3 s to give a clearer view. The figure shows clearly 807 that, on the whole, the total sound field with the potential-808 based control sources resembles closely the original wanted 809 sound fields at each measuring position in the shielded do-810 main. The similarity of the net sound field shielded by the 811 AS control sources and the original wanted sound fields is 812 also evaluated by the cross-correlation of the two time signa-813 tures<sup>29</sup> shown in Fig. 7. The cross-correlation of the wanted 814 sound and the total sound pressure that consists of the 815 unwanted noise and the wanted sounds without controls is at 816 a maximum of 0.67 at zero time-lag. When the AS control 817 sources are switched on, the cross-correlation of the wanted 818 sound and the shielded total sound pressure (the unwanted 819 noise, the wanted sounds, and the controls together) jumps to 820



FIG. 6. In a scale of one-third octave bands: (a) sound pressure distribution (overall noise about 10 dB higher than the wanted sound); O: wanted sound pressure, sure, Δ: shielded total sound pressure (the sum of unwanted, wanted sound, and control outputs), \*: unwanted sound pressure, and ×: the sum of unwanted and wanted sound pressure, and (b) errors between actively shielded results and the wanted sound against the difference between unwanted and wanted sound pressure levels.
837
838
839
840

J. Acoust. Soc. Am., Vol. 129, No. 2, February 2011



FIG. 7. Sound pressure in time domain: (a) the sound pressure of noise and wanted sound without control, (b) shielded total sound pressure (noise, wanted sounds, and controls), (c) wanted sound pressure, and (d) **E E**: wanted sound pressure, and  $\bigcirc$ : shielded total sound pressure.

0.99 at the same point of time-lag. The ideal cross-correlation of two identical signals is 1.0. This is almost achieved in the experiment, which shows that the shielded net sound field with the controls on matches the original wanted sound field very well. The experiment clearly proves that wanted sounds can be very effectively protected by the active controls based on the proposed method even in multi-domains where wanted sounds from different shielded domains can also interfere with each other, while unwanted noise is significantly suppressed by the AS control sources.

Despite the difficulty in dealing with multiple control sources and the complexity of multi-domains, the results in this section show better efficiency in the overall attenuation and preservation of the wanted sounds, when compared with the result discussed in the previous section. One of the main reasons can be found in the design of the experimental model. That is, the terminations are more absorptive in this multi-domain experiment, which damped the tube resonan-ces. At resonances and anti-resonances the result is much more sensitive to measurement errors due to large changes in the sound pressure,<sup>28</sup> see also the SA in Sec. III B. By damping out the resonances, this sensitivity to error is signif-icantly reduced. The other reason for the better result is that the accuracy of the control system has also been improved by explicitly taking some of the causes of inherent measure-

ment errors into account. For instance, the phase distortion caused by a time delay in the digital-to-analog converter used is corrected for in this latest measurement. According to the SA in Sec. III B, it has been shown that the control system is most sensitive to time delay errors. Correcting for this error thus improves the result significantly. All these together make a significant contribution to the stability of the whole system and the repeatability of the measurements.

### **IV. CONCLUSIONS**

The practicality of AS based on the method of differ-ence potentials has been experimentally validated with broadband acoustic sources in a variety of one-dimensional bounded domains. Unlike previous experiments with pure tone sources, the sound fields generated by broadband sig-nals in the experiments presented in this paper include sub-stantial resonances and anti-resonances in the system. Through these experiments, it has been shown that attenua-tion from 15 to 20 dB can be achieved even at resonances in the shielded domains, whether or not wanted sounds exist in the same space. Similar to other existing AS methods, the effectiveness of the control solution based on the difference potentials method are most sensitive to time delay errors especially at resonances. Apart from the practical difficulties 

993

994

995

996

999

1009

1010

1011

1012

1013

1014

1015

1016

1017

1018

1019

1020

AQ1997 998 associated with this high sensitivity at resonances, the present experiments show that the method can provide an effective solution not only at non-resonance regimes but also through a full continuous broadband spectrum of frequency that includes resonances. However, in practice, it is difficult to find an accurate solution at strong anti-resonances when the physical values of the total sound field to be measured at the boundary are too small to be measured with sufficient accuracy. This is limited by the dynamic range of the measurement probe and the very low signal to noise ratio at strong anti-resonances.

The level of attenuation found in the result of the experiments is similar to the those achieved by other existing conventional AS methods when broadband sources are used in a one-dimensional enclosure. In addition to the significant suppression of noise, the proposed method has been shown to also effectively preserve the wanted sounds separately from the total fields of noise and wanted sounds, even when there are multiple shielded domains with interfering wanted sounds from different domains, and that the system characteristics are not known. The results clearly demonstrate the potential advantages of the method in practical applications under these conditions.

This paper has shown that the proposed approach can be realized provided that the contribution of the control sources to the input data can be separated. An obvious question to follow up is how to obtain such separation in practice. Theoretically, it has been proven in Ref. 18 that this can be done via a modification of the solution presented by Eq. (6). It requires additional on-line calculations of surface potentials, which can be efficiently carried out via the method of difference potentials. However, the implementation of the algorithm is far from trivial and will be an objective of our future research.

### ACKNOWLEDGMENTS

The research was supported by the EPSRC under the project codes, GR/T26825/01 and GR/T26832/01.

- <sup>1</sup>P. A. Nelson and S. J. Elliott, *Active Control of Sound* (Academic Press, San Diego, CA, 1992), pp. 116–122, 143–146, 311–378.
- <sup>2</sup>R. K. Kincaid, S. L. Padula, and D. L. Palumbo, "Optimal sensor/actuator locations for active structural acoustic control," AIAA Paper 98-1865, in *Proceedings of the 39th AIAA/ASME/ASCE/AHS/ASC Structures, Dynamics and Materials Conference*, Long Beach, CA (1998).
- <sup>3</sup>R. K. Kincaid and K. Laba, "Reactive tabu search and sensor selection in active structural control problems," J. Heuristics 4, 199–220 (1998).
- <sup>4</sup>M. Hodgson, J. Guo, and P. Germain, "Active local control of propelleraircraft run-up noise," J. Acoust. Soc. Am. **114**(6), 3201–3210 (2003).

- <sup>5</sup>V. S. Ryaben'kii, "A difference shielding problem," J. Funct. Anal. Appl. 29(1), 70–71, 1995.
   <sup>6</sup>V. S. Ryaben'kii, *Method of Difference Potentials and its Applications* (Springer-Verlag, Berlin, 2002), pp. 515–522.
   <sup>7</sup>M. J. M. Jessel and G. A. Mangiante, "Active sound absorbers in an air 1025
- duct," J. Sound Vib. **23**(3), 383–390, 1972. <sup>8</sup>G. A. Mangiante, "Active sound absorption," J. Acoust. Soc. Am. **61**(6), 1026
- 1027 C carry t "Active sound absorption", J. Actust. Soc. Am. 01(0), 1027
- <sup>9</sup>G. Canevet, "Active sound absorption in air conditioning duct," J. Sound Vib. **58**(3), 333–345, 1978.
- <sup>10</sup>G. Mangiante, "The JMC Method for 3D active sound absorption: a numerical simulation," Noise Control Eng. J. **41**(2), 339–345, 1993.
- <sup>11</sup>V. S. Ryaben'kii, S. V. Tsynkov, and S. V. Utyuzhnikov, "Inverse source problem and active shielding for composite domains," J. Appl. Math. Lett. **20**(5), 511–515, 2007.
   <sup>10</sup>A W. Paterson and S. V. Tsynkov, "Active central of cound for comparise."
- <sup>12</sup>A. W. Peterson and S. V. Tsynkov, "Active control of sound for composite regions," SIAM J. **67**(6), 1582–1609, 2007. 1034
- <sup>13</sup>J. Loncaric, V. S. Ryaben'kii, and S. V. Tsynkov, "Active shielding and control of noise," SIAM J. **62**(2), 563–596, 2001. 1036
- <sup>14</sup>S. V. Tsynkov, "On the definition of surface potentials for finite-difference operators," J. Sci. Comput. 18, 155–189, 2003.
- <sup>15</sup>V. S. Ryaben'kii, S. V. Utyuzhnikov, and A. Turan, "On the application of difference potential theory to active noise control," J. Adv. Appl. Math. 40(2), 194–211, 2008.
- 40(2), 194–211, 2008.
   <sup>16</sup>V. S. Ryaben'kii, and S. V. Utyuzhnikov, "Active shielding model for hyperbolic equations," IMA J. Appl. Math. 71(6), 924–939, 2006.
- <sup>17</sup>Yu. V. Egorov and M. A. Shubin, *Foundations of the Classical Theory of Partial Differential Equations* (Springer, Berlin-London, 1992). 1043
- <sup>18</sup>S. V. Utyuzhnikov, "Generalized Calderon-Ryaben'kii's potentials," IMA
   J. Appl. Math. 74(1), 128–148, 2009.
   <sup>19</sup>S. V. Utyuzhnikov, "Active wave control and generalized surface 1045
- <sup>19</sup>S. V. Utyuzhnikov, "Active wave control and generalized surface potentials," J. Adv. Appl. Math. 43(2), 101–112, 2009. 1046
- <sup>20</sup>H. Lim, S. V. Utyuzhnikov, Y. W. Lam, A. Turan, M. R. Avis, V. S. Ryaben'kii, and S. V. Tsynkov, "Experimental validation of the active noise control methodology based on difference potentials," AIAA J. **47**(4), 874– 884, 2009.
- <sup>21</sup>J. Loncaric and S. V. Tsynkov, "Optimization of acoustic source strength in the problems of active noise control," SIAM J. Appl. Math. **63**, 1141– 1183, 2003.
- <sup>22</sup>V. S. Ryaben'kii, S. V. Tsynkov, and S. V. Utyuzhnikov, "Active control of sound with variable degree of cancellation," J. Appl. Math. Lett. 22(12), 1846–1851, 2009.
- <sup>23</sup>A. J. Zuckerwar and R. W. Meredith, "Radiation losses in resonant tubes,"
   J. Acoust. Soc. Am. 70, 879–885, 1981.
- <sup>24</sup>C. H. Hansen and S. D. Snyder, *Active Control of Noise and Vibration*, (E. & F. N. Spon, London, 1997), pp. 555–556.
- <sup>25</sup>S. Uosukainen and V. Valimaki, *JMC Actuators and Their Applications in Active Attenuation of Noise in Ducts (VTT Publications*, 341, VTT Building Technology, ESPOO, 1998), p. 100.
   <sup>26</sup>S. Losukainen "Modified IMC method in active control of cound." Acust
- <sup>26</sup>S. Uosukainen, "Modified JMC method in active control of sound," Acust. Acta Acust. 83, 105–112, 1997.
- <sup>27</sup>K. Anai, T. Shiki, Y. Hiraguri, and K. Fujimoto, "Improving sound insulation capability at a ventilation opening using active noise control: Improving sound insulation performance degraded by living sound," in *Proceedings of Inter-Noise 2008*, Shanghai, Paper No.0252 (2008).
- <sup>28</sup>S. V. Utyuzhnikov, "Nonstationary problem of active sound control in bounded domains," J. Comput. Appl. Math. 234(6), 1725–1731, 2010.
- <sup>29</sup>R. B. Randall and B. Tech, *Frequency Analysis* (Brüel& Kjær, Denmark, 1987), pp. 252–253.
   1068
  - 1073 1074

1069

1070

1071

1072

1037

1047

1048

1049

1050

1051

1052

1053

1054

1055

1056

1057

1061

- 1075 1076
- 1070
- 1077
- 1079
- 1080

J. Acoust. Soc. Am., Vol. 129, No. 2, February 2011

AQ1: Please provide the definition of "EPSRC" in the acknowledgment.